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# An exchange-gas vibration Isolation system for a general-purpose 4K research cryostat

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# An exchange-gas vibration isolation system for a generalpurpose 4K research cryostat

# Motivation

- Sensitive SQUID sensors
- ADR temperature < 50mK
- Future: ultralow temperatures and vibration
- General purpose research cryostat
- Large 4K experimental space (35 liters)
- 45A superconducting magnet, iron and lead shielding (20 kg)
- Cryogen free
  - Pulse-tube refrigerator
  - High power dissipation at 65K
  - ➢ Good thermal coupling at 4K

# **Vibration isolation approaches**





- Goal: Gas impedance better than PTR source impedance
- Model as closed parallelepiped with vertical hot and cold surfaces



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## Implementation

•Cryomech Inc PTR-405 pulse tube refrigerator •OFHC Cu heat exchangers •Gas Gap: 1.47 mm •XGV envelope made from G-10 tubes Stage 2 Stage 1 Area: 1038 cm<sup>2</sup> Area: 580 cm<sup>2</sup>



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# Instrumentation

#### Thermal management

- Cernox and Si-diode temperature sensors
- 25 W heater on stage 1 (external)
- 1 W heater on stage 2 (external)
- Software PI control

#### **Exchange gas control**

- MKS Baratron 626, 1000 Torr
- Automated fill- and pump-valves
- Pressure regulated within 3 Torr



# **Thermal Performance Study**

- Measurement process:
  - Regulate XGV pressure & stage 1 and 2 *external* temperatures
  - Record external and internal temperatures & servo powers



Extract equilibrium heater powers and thermal standoffs

## **Thermal Performance Study**

XGV pressure [Torr 400 200 Measurement process: 18:00 00:00 12:00 06:00 18:00 Regulate XGV pressure & stage 1 Stage 2 external and 2 external temperatures Σ internal Record external and internal 18:00 00:00 06:00 12:00 18:00 temperatures & servo powers Heater Power [W] 0.5 Data reduction 18:00 00:00 06:00 12:00 18:00 14 Gas Gas O 1.6 fit 0.30 K/W fit 0.71 K/W 12 PTR Stage 1 PTR Stage 2 1.4 fit 1.84 K/W fit 0.98 K/W 10 Stage 1 ∆ T [K] Stage 2 ∆ T [K] 1.2 8 0.8 6 0.6 0.4 **Heat load** 2 0.2 **Heat load** ~11 W ~150mW 0 0 10 12 14 0.2 0.4 0.6 0.8 1.2 0 2 4 6 8 0 1 Stage 1 heater power [W] Stage 2 heater power [W]

#### **Gas Thermal Impedance**



#### **External Temperatures versus Heat Load**

Stage 1

Stage 2



T < 4.8K for 0.5W

## **Running a 45A Magnet**

- Hi-T<sub>c</sub> current leads: HTS-110 Cryosaver, 100A
- Brass leads from room temperature to stage 1

optimized for lowest power at 50A

Joule heating at 45A: 3.25W Thermal impedance: 1.0 K/W Heat leak through brass: 4.93 W



# **Vibration measurements**



- three HS-1 geophones
- low noise preamplifier
- PTR separately supported from roof
- XGV under slight overpressure













## **Vibration isolation**



# Summary

- Exchange-gas approach appears feasible for larger generalpurpose cryostats
  - Thermal impedance agrees with design calculations
  - Preserves cooling power and base temperature well
    - Good conductance through exchange gas
    - No evidence of Taconis oscillations or convection rolls
- Promising initial vibration isolation results
  - PTR noise already negligible compared to floor noise
  - Clear path to further improvement:
    - Increase stiffness of PTR support plate
    - Increase damping of cryostat floating table's eigenmodes
- E-mail for manuscript: stpboyd@unm.edu

## **Thermal Impedance Estimate**



High T, small Ra, laminar boundary layer -> diffusive Low T, large Ra, turbulent convection

At 65K, gap matters moreAt 4K, convection boosts 5-fold